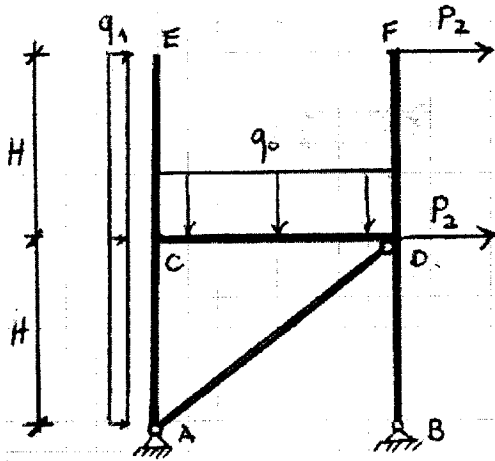


DATI.



GEOMETRIA:  
 $L = 400 \text{ cm}$   
 $H = 300 \text{ cm}$

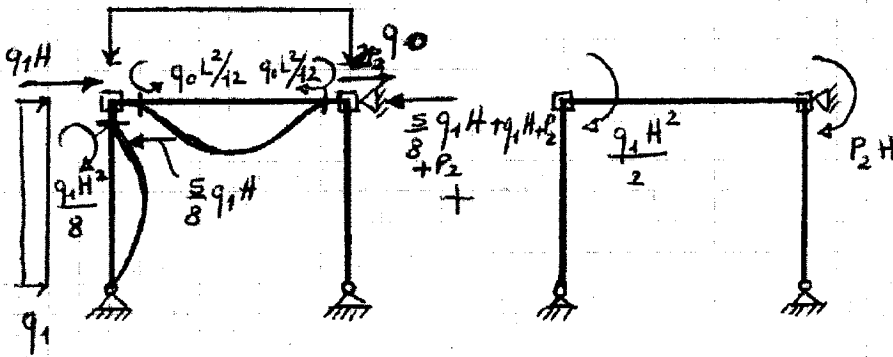
CARICHI:  
 $q_1 = 500 \frac{\text{Kg}}{\text{m}} = 5 \frac{\text{Kg}}{\text{cm}}$   
 $q_0 = 2000 \frac{\text{Kg}}{\text{m}} = 20 \frac{\text{Kg}}{\text{cm}}$

MATERIALI  
 $E = 210000 \frac{\text{Kg}}{\text{cm}^2}$   
 $G_{am} = 2400 \frac{\text{Kg}}{\text{cm}^2}$

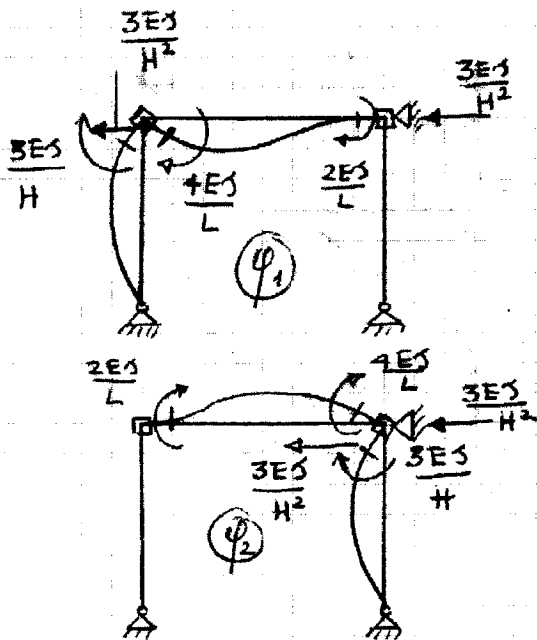
$P_2 = 1000 \text{ Kg}$



A1) PROGETTO: TRAVI EF E FD ISOSTATICHE



CARICHI NODALI



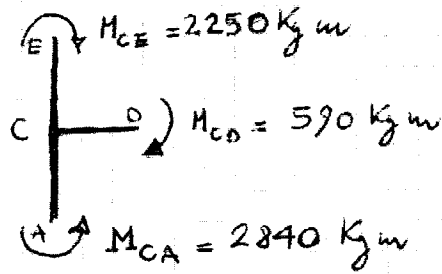
$$\begin{bmatrix} \frac{4}{L} + \frac{3}{H} & \frac{2}{L} \\ \frac{2}{L} & \frac{4}{L} + \frac{3}{H} \end{bmatrix} \begin{bmatrix} ES\varphi_1 \\ ES\varphi_2 \end{bmatrix} = \begin{bmatrix} \frac{q_0 L^2}{12} + \frac{q_1 H^2}{2} - \frac{q_1 H^2}{8} \\ -\frac{q_0 L^2}{12} + P_2 H \end{bmatrix}$$

$$\Rightarrow \begin{cases} ES\varphi_1 = 2,27 \cdot 10^7 \text{ Kg cm}^2 \\ ES\varphi_2 = -4,027 \cdot 10^6 \text{ Kg cm}^2 \end{cases}$$

$$\curvearrowleft M_{CA} = \frac{3}{H} ES \varphi_1 + \frac{q_1 H^2}{8} = 284028 \text{ Kg cm} = 2840 \text{ Kg m}$$

$$\curvearrowleft M_{CO} = \frac{4}{L} ES \varphi_1 + \frac{2}{L} ES \varphi_2 - \frac{q_0 L^2}{12} = -59028 \text{ Kg cm} = -590 \text{ Kg m}$$

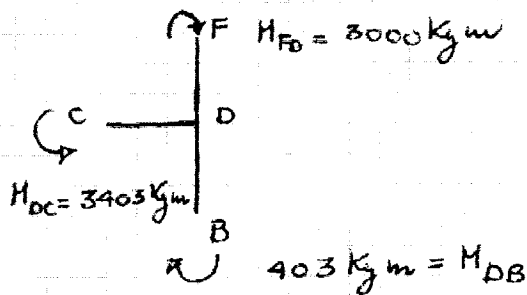
$$\curvearrowleft M_{CE} = -\frac{q_1 H^2}{2} = -225000 \text{ Kg cm} = -2250 \text{ Kg m}$$



$$\curvearrowleft M_{DB} = \frac{3}{H} ES \varphi_2 = -40277.7 \text{ Kg cm} = -403 \text{ Kg m}$$

$$\curvearrowleft M_{DC} = \frac{4}{L} ES \varphi_2 + \frac{2}{L} ES \varphi_1 + \frac{q_0 L^2}{12} = 340278 \text{ Kg cm} = 3403 \text{ Kg m}$$

$$\curvearrowleft M_{FD} = -P_2 H = -300000 \text{ Kg cm} = -3000 \text{ Kg m}$$



PROGETTO:

$$M_{max} = M_{DC} = 340278 \text{ Kg cm}$$

$$W_{min} = \frac{M_{max}}{6 \sigma_m} = 142 \text{ cm}^3 \rightarrow \text{HEA 140}$$

A	W	I
31,4	155	1033
cm <sup>2</sup>	cm <sup>3</sup>	cm <sup>4</sup>

$$\leftarrow T_{AC} = \frac{3}{8} q_1 H - \frac{3}{H^2} ES \varphi_1 = -197 \text{ Kg}$$

$$\leftarrow T_{CA} = \frac{3}{H^2} ES \varphi_1 + \frac{5}{8} q_1 H = 1697 \text{ Kg}$$

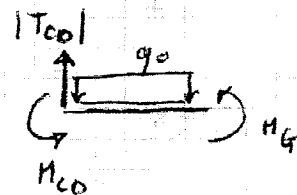
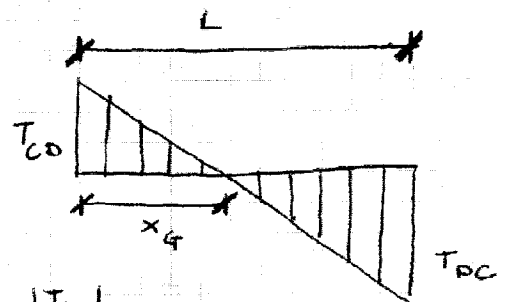
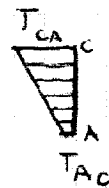
$$\leftarrow T_{EC} = q_1 H = 1500 \text{ Kg}$$

$$\downarrow T_{CO} = \frac{6}{L^2} ES \varphi_1 + \frac{6}{L^2} ES \varphi_2 - \frac{q_0 L}{2} = -3297 \text{ Kg}$$

$$\uparrow T_{OC} = \frac{6}{L^2} ES \varphi_1 + \frac{6}{L^2} ES \varphi_2 + \frac{q_0 L}{2} = 4703 \text{ Kg}$$

$$\leftarrow T_{FD} = P_2 = 1000 \text{ Kg}$$

$$\leftarrow T_{DB} = \frac{3}{H^2} ES \varphi_2 = -134 \text{ Kg}$$



MASSIMO MOMENTO IN CAMPATA  $\overline{CO}$ :

$$x_g = \frac{|T_{CO}|}{|T_{CO}| + |T_{OC}|} L = 164,85 \text{ cm}$$

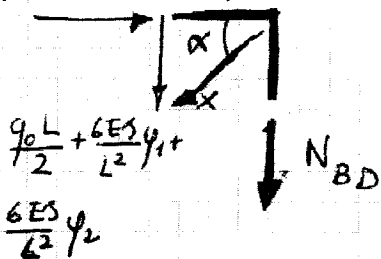
$$M_g = -M_{CO} - \frac{q_0 x_g^2}{2} + T_{CO} x_g = 212727 \text{ Kg cm}$$

$$= 2127 \text{ Kg m}$$

$$N_{CA} = -\frac{q_0 L}{2} + \frac{6ES}{L^2} \psi_1 + \frac{6ES}{L^2} \psi_2 = -3297 \text{ Kg}$$

$$N_{CD} = \frac{5}{8} q_1 H + q_1 H + \frac{3}{H^2} ES \psi_1 = -3197 \text{ Kg}$$

$$\frac{5}{8} q_1 H + q_1 H + 2P_2 + \frac{3}{H^2} ES \psi_1 + \frac{3}{H^2} ES \psi_2$$



$$\tan \alpha = \frac{H}{L}$$

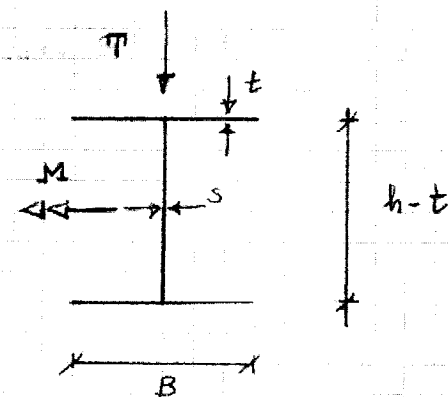
$$X = \left( \frac{5}{8} q_1 H + q_1 H + 2P_2 + \frac{3}{H^2} ES \psi_1 + \frac{3}{H^2} ES \psi_2 \right) \cos \alpha = 6328 \text{ Kg}$$

$$N_{BD} = -X \sin \alpha - \left[ \frac{q_0 L}{2} + \frac{6}{L^2} (ES \psi_1 + ES \psi_2) \right] = -8500 \text{ Kg}$$

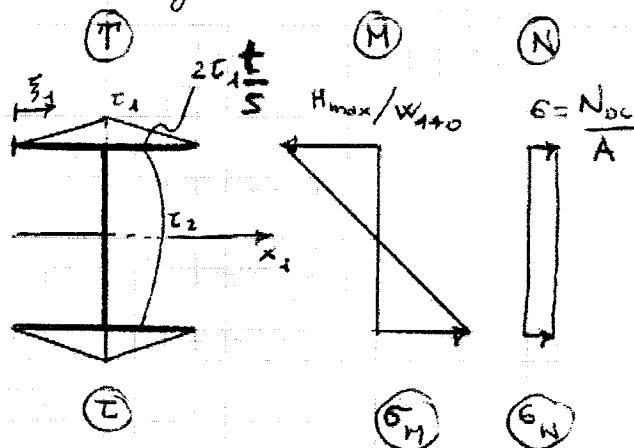
$T_{DC}$

A2) VERIFICA DELLO STATO TENSIONALE NELLA SEZIONE MAGGIORMENTE SOLLECITATA

$$M_{DC} = 340278 \text{ Kg} \cdot \text{cm} \quad T_{DC} = 4703 \text{ Kg} \quad N_{DC} = -3197 \text{ Kg}$$



$$\begin{aligned} h &= 133 \text{ cm} \\ B &= 14 \text{ cm} \\ t &= 0,85 \text{ cm} \\ s &= 0,55 \text{ cm} \end{aligned}$$



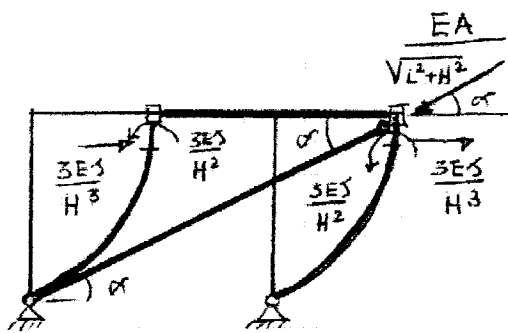
$$\sigma_{id} = \sqrt{\sigma^2 + 3\tau_1^2} = 2323 \frac{\text{Kg}}{\text{cm}^2} < \sigma_{cm} \rightarrow \text{VERIFICATO}$$

$$\tau_1 = \frac{T \cdot \frac{B}{2} \cdot \frac{h-t}{2}}{\frac{B}{2} \cdot \frac{h-t}{2}} = \frac{T(h-t)B}{4s}$$

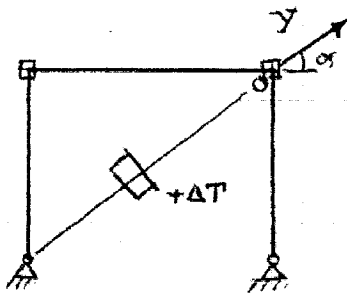
$$\tau_2 = \frac{T \left[ \frac{B(h-t)t}{2} + s \frac{(h-t)}{2} \cdot \frac{(h-t)}{4} \right]}{s s} = 701 \frac{\text{Kg}}{\text{cm}^2}$$

$$\sigma = \sigma_H + \sigma_N$$

B)



(u)



$$-\frac{Y}{EA} \sqrt{L^2 + H^2} + \alpha \Delta T \sqrt{L^2 + H^2} = 0$$

$$Y = \alpha \Delta T EA - \Delta T (\alpha EA)$$

$$\frac{A}{J} = \frac{5 \text{ cm}^2}{1033 \text{ cm}^4} = 0,00484$$

$$\alpha EA = 1,2 \cdot 10^{-5} \cdot 2100 \cdot 000 \cdot 5$$

$$\begin{bmatrix} \frac{4}{L} + \frac{3}{H} & \frac{2}{L} & -\frac{3}{H^2} \\ \frac{2}{L} & \frac{4}{L} + \frac{3}{H} & -\frac{3}{H^2} \\ -\frac{3}{H^2} & -\frac{3}{H^2} & \frac{6}{H^3} + \frac{A}{J} \frac{\cos \alpha}{\sqrt{L^2 + H^2}} \end{bmatrix} \begin{bmatrix} ES \varphi_1 \\ ES \varphi_2 \\ ES u \end{bmatrix} = \begin{bmatrix} \frac{q_0 L^2}{12} + \frac{q_1 H^2}{2} - \frac{q_1 H^2}{8} \\ -\frac{q_0 L^2}{12} + P_2 H \\ \frac{5}{8} q_1 H + q_1 H + 2 \frac{P_2}{2} + X_{\text{geo}} \end{bmatrix}$$

$$\Rightarrow \begin{cases} ES \varphi_1 = 2,49061 \cdot 10^7 \text{ Kg cm}^2 & \varphi_1 = 0,011481 \text{ rad} \\ ES \varphi_2 = -1,89944 \cdot 10^6 \text{ Kg cm}^2 & \varphi_2 = -0,000876 \text{ rad} \\ ES u = 1,59625 \cdot 10^8 \text{ Kg cm}^3 & \Rightarrow u = 0,735838 \text{ cm} \end{cases}$$

MOMENTI:

$$M_{CA} = \frac{3}{H} ES \varphi_1 - \frac{3}{H^2} ES u + \frac{q_1 H^2}{8} = 252103 \text{ Kg cm} = 2521 \text{ Kg m}$$

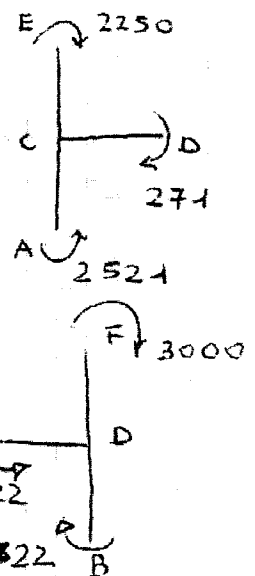
$$M_{CD} = \frac{4}{L} ES \varphi_1 + \frac{2}{L} ES \varphi_2 - \frac{q_0 L^2}{12} = -27103 \text{ Kg cm} = -271 \text{ Kg m}$$

$$M_{CE} = -225000 \text{ Kg cm} = -2250 \text{ Kg m}$$

$$M_{DB} = \frac{3}{H} ES \varphi_2 - \frac{3}{H^2} ES u = -72203 \text{ Kg cm} = -722 \text{ Kg m}$$

$$M_{DC} = \frac{4}{L} ES \varphi_2 + \frac{2}{L} ES \varphi_1 + \frac{q_0 L^2}{12} = 372203 \text{ Kg cm} = 3722 \text{ Kg m}$$

$$M_{FD} = -3000 \text{ Kg m}$$



TAGLI:

$$-T_{AC} = \frac{3}{8} q_1 H - \frac{3}{H^2} ES \varphi_1 + \frac{3}{H^3} ES u = -20,3 \text{ Ky}$$

$$-T_{CA} = \frac{3}{H^2} ES \varphi_1 - \frac{3}{H^3} ES u + \frac{5}{8} q_1 H = 1590 \text{ Ky}$$

$$-T_{EC} = 1500 \text{ Ky}$$

$$\downarrow T_{CD} = \frac{6}{L^2} ES \varphi_1 + \frac{6}{L^2} ES \varphi_2 - \frac{q_0 L}{2} = -3137 \text{ Ky}$$

$$\uparrow T_{DC} = \frac{6}{L^2} ES \varphi_1 + \frac{6}{L^2} ES \varphi_2 + \frac{q_0 L}{2} = 4863 \text{ Ky}$$

$$-T_{FD} = 1000 \text{ Ky}$$

$$-T_{DB} = \frac{3}{H^2} ES \varphi_2 - \frac{3}{H^3} ES u = -241 \text{ Ky}$$

MASSIMO MOMENTO IN CAMPATA  $\bar{C}O$ :

$$x_g = \frac{|T_{CO}|}{|T_{CO}| + |T_{DC}|} L = 156,86 \text{ cm}$$

$$M_g = -M_{CO} - \frac{q_0 x_g^2}{2} + T_{CO} x_g = 218919 \text{ Ky cm} = 2189 \text{ Ky m}$$

SFORZI NORMALI:

$$\uparrow N_{CA} = -\frac{q_0 L}{2} + \frac{6ES}{L^2} \varphi_1 + \frac{6ES}{L^2} \varphi_2 = -3137 \text{ Ky}$$

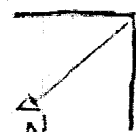
$$-N_{CD} = \frac{5}{8} q_1 H + q_1 H + \frac{3}{H^2} ES \varphi_1 - \frac{3}{H^3} ES u = 3090 \text{ Ky}$$

$$N_{AD} = \frac{A \cos \alpha}{\sqrt{L^2 + H^2}} ES u = 6062 \text{ Ky}$$

$$\uparrow N_{DB} = N_{AD} \sin \alpha + \frac{6}{L^2} ES \varphi_1 + \frac{6}{L^2} ES \varphi_2 + \frac{q_0 L}{2} = 8500 \text{ Ky}$$

$T_{DC}$

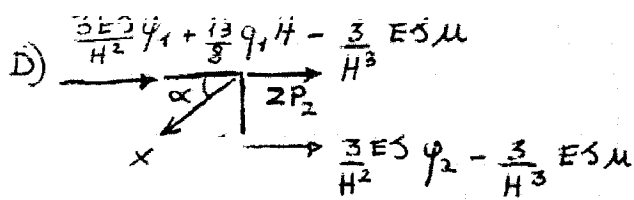
$$T_{DC} = \frac{6}{L^2} ES \varphi_1 + \frac{6}{L^2} ES \varphi_2 + \frac{q_0 L}{2}$$



c) SPOSTAMENTO NODI D e F:

$$u_D = u = 0,735838 \text{ cm}$$

$$u_F = u + \frac{P_2 H^3}{3ES} + \varphi_2 H = 4,622 \text{ cm}$$

D) 

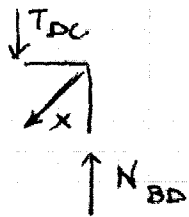
$$X \cos \alpha = \frac{3}{H^2} ES \psi_1 + \frac{3}{H^2} ES \psi_2 - \frac{6}{H^3} ES \mu + 2P_2 + \frac{13}{8} q_1 H$$

$$= 0$$

$$\begin{array}{l} 1) \left[ \begin{array}{ccc} \frac{4}{L} + \frac{3}{H} & \frac{2}{L} & -\frac{3}{H^2} \\ \frac{2}{L} & \frac{4}{L} + \frac{3}{H} & -\frac{3}{H^2} \\ -\frac{3}{H^2} & -\frac{3}{H^2} & \frac{6}{H^3} + \frac{A}{S} \frac{\cos^3 \alpha}{\sqrt{L^2 + H^2}} \\ -\frac{3}{H^2} & -\frac{3}{H^2} & +\frac{6}{H^3} \end{array} \right] \begin{bmatrix} ES \psi_1 \\ ES \psi_2 \\ \mu \\ \Delta T \end{bmatrix} = \begin{bmatrix} \frac{q_0 L^2}{12} + \frac{q_1 H^2}{2} - \frac{q_1 H^2}{8} \\ -\frac{q_0 L^2}{12} + P_2 H \\ \frac{5}{8} q_1 H + q_1 H + 2P_2 \\ 2P_2 + \frac{13}{8} q_1 H \end{bmatrix} \\ 2) \\ 3) \\ 4) \end{array}$$

$$\Rightarrow \Delta T = 2.3337^\circ C$$

N.B.:



$N_{BD}$  E' CALCOLABILE PER EQ. ALLA ROTAZIONE RISPETTO AD A DELL'INTERA STRUTTURA

$$N_{BD} = \frac{P_2 3H + q_1 2H^2 + q_0 L^2/2}{L} = 8500 \text{ kg}$$

$$T_{DC} = \frac{6}{L^2} ES \psi_1 + \frac{6}{L^2} ES \psi_2 + \frac{q_0 L}{2}$$

$$X \sin \alpha = N_{BD} - T_{DC} = \frac{3P_2 H}{L} + 2q_1 \frac{H^2}{L} - \frac{6}{L^2} ES \psi_1 - \frac{6}{L^2} ES \psi_2$$

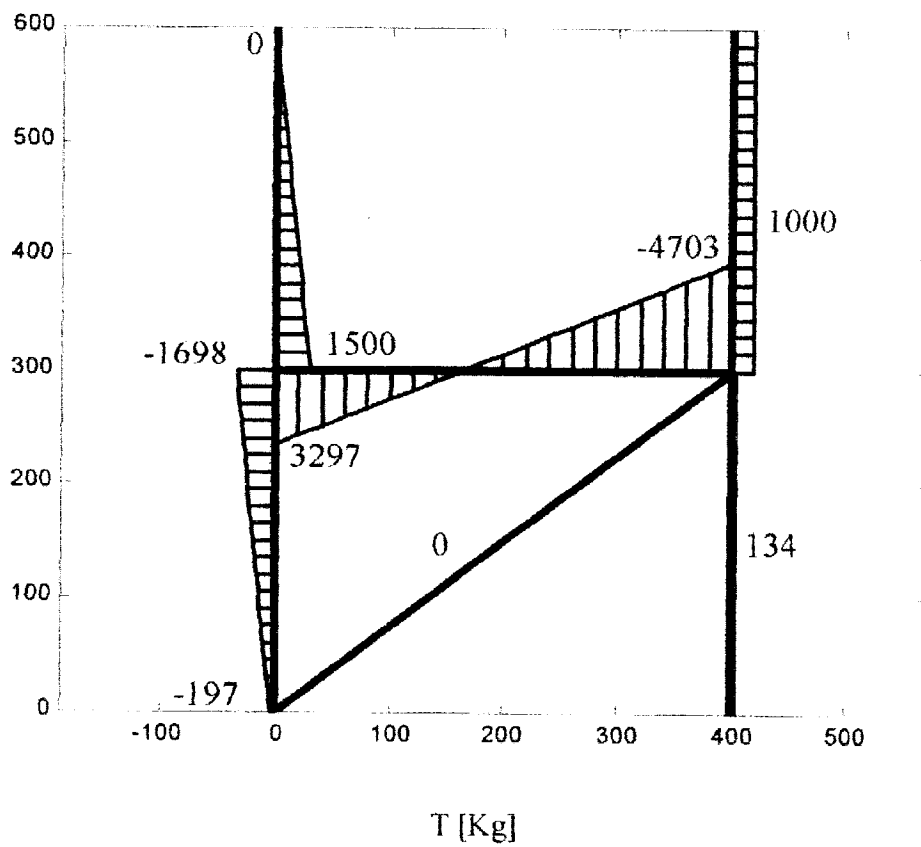
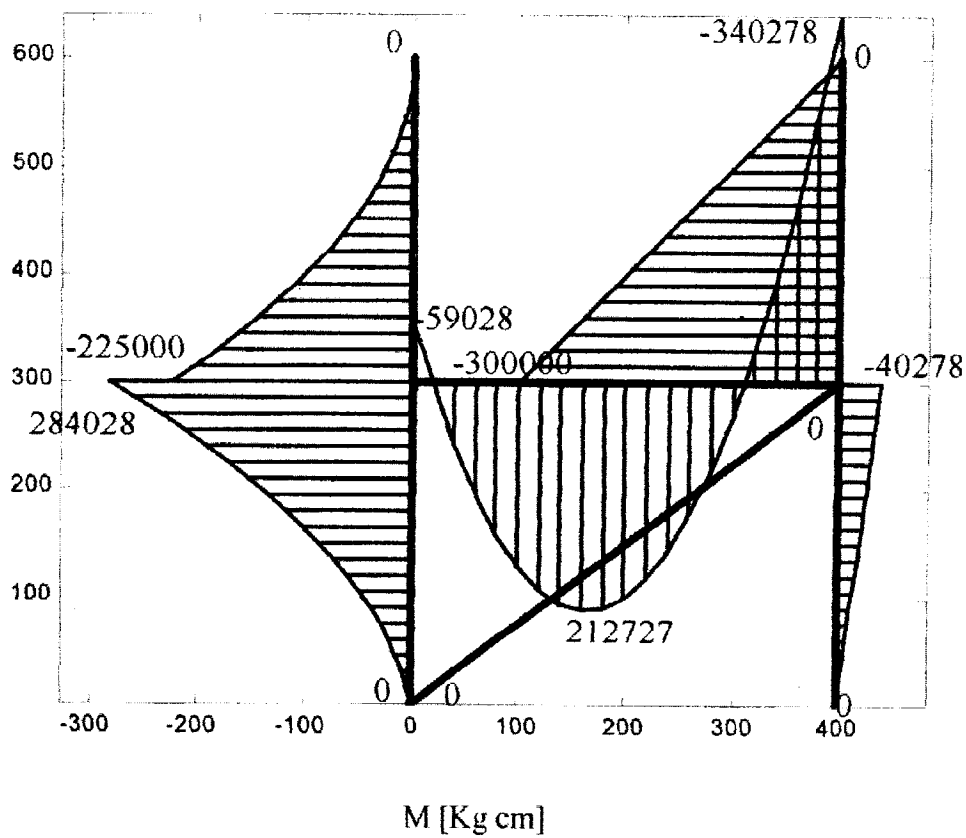
$$= 0 \quad (*)$$

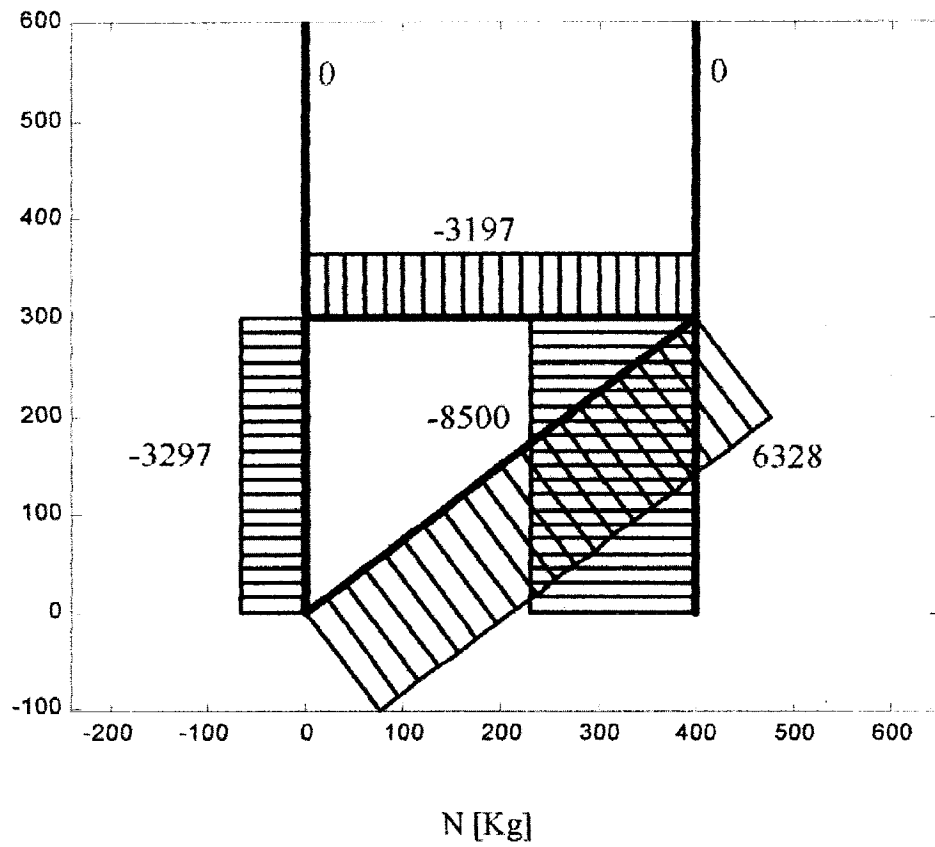
SOMMANDO ~~NON~~ 1) c2) A (\*) [MOLTIPLICATA PER L] SI OTTIENE LA  $\dagger$  [MOLTIPLICATA PER H]

$$-\frac{6}{H^2} \mu + \left(\frac{6}{L} + \frac{3}{H}\right) ES \psi_1 + \left(\frac{6}{L} + \frac{3}{H}\right) ES \psi_2 - \frac{3}{8} q_1 H^2 - P_2 H + 3P_2 \frac{H}{2} + 2q_1 \frac{H^2}{4} - \frac{6}{L^2} ES \psi_1 - \frac{6}{L^2} ES \psi_2 = 0$$

$$\Rightarrow -\frac{3}{H} ES \psi_1 - \frac{3}{H} ES \psi_2 + \frac{6}{H^2} \mu = \frac{13}{8} q_1 H^2 + 2P_2 H = 0$$

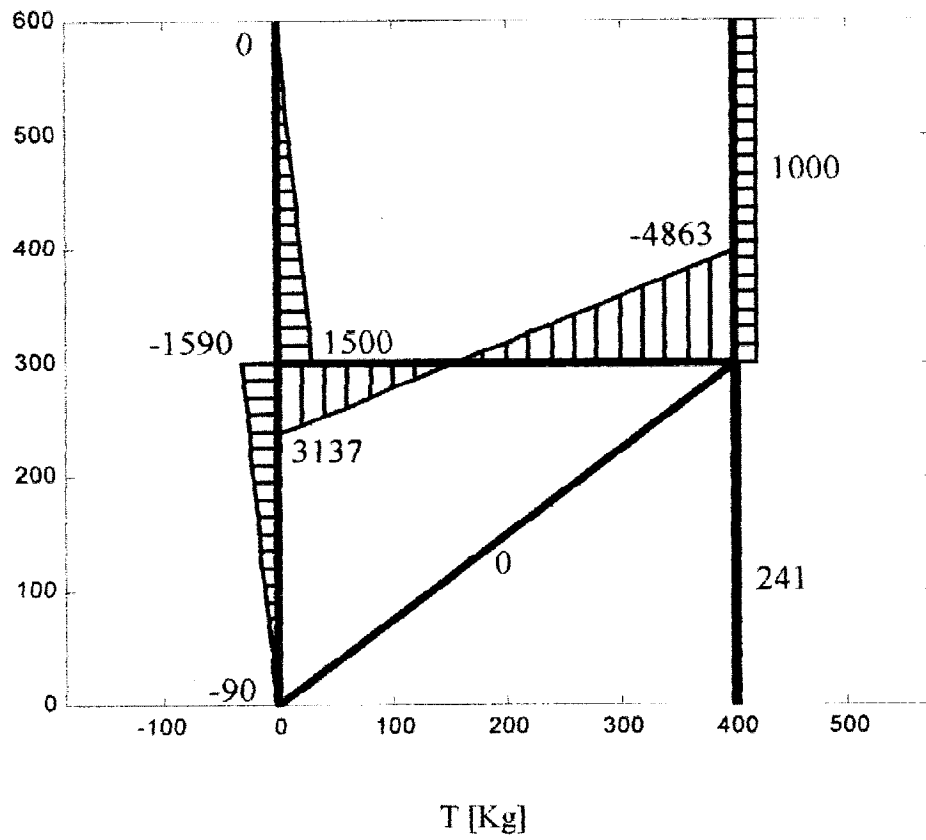
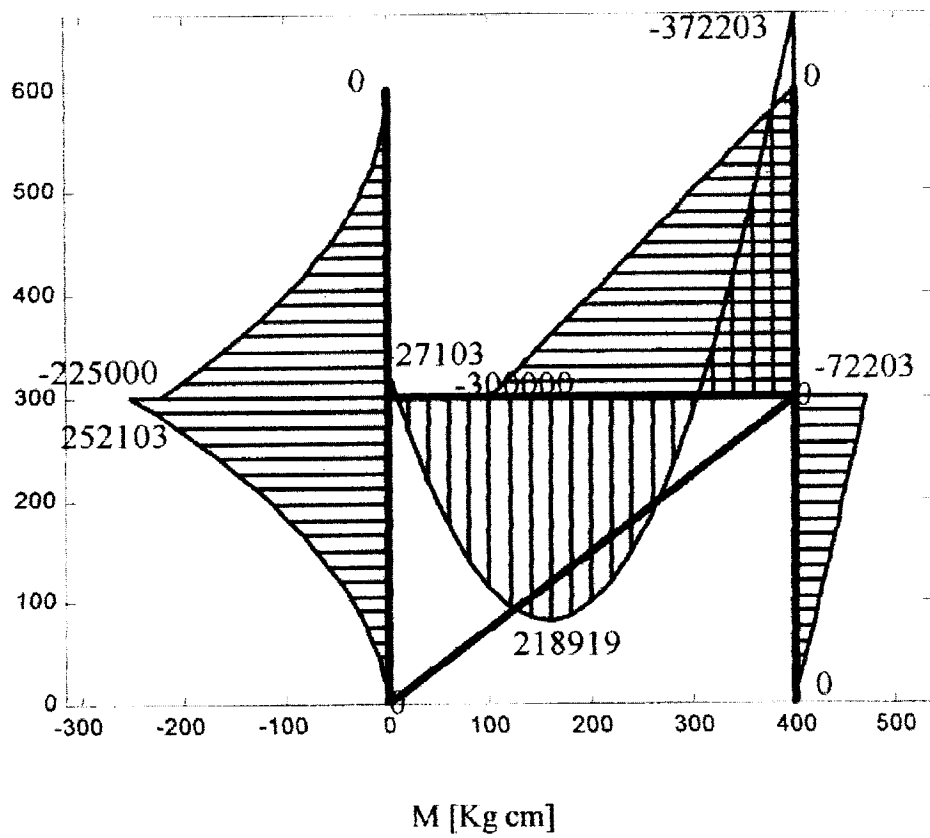
$$\Rightarrow -\frac{3}{H^2} ES \psi_1 - \frac{3}{H^2} ES \psi_2 + \frac{6}{H^3} \mu = \frac{13}{8} q_1 H + 2P_2$$



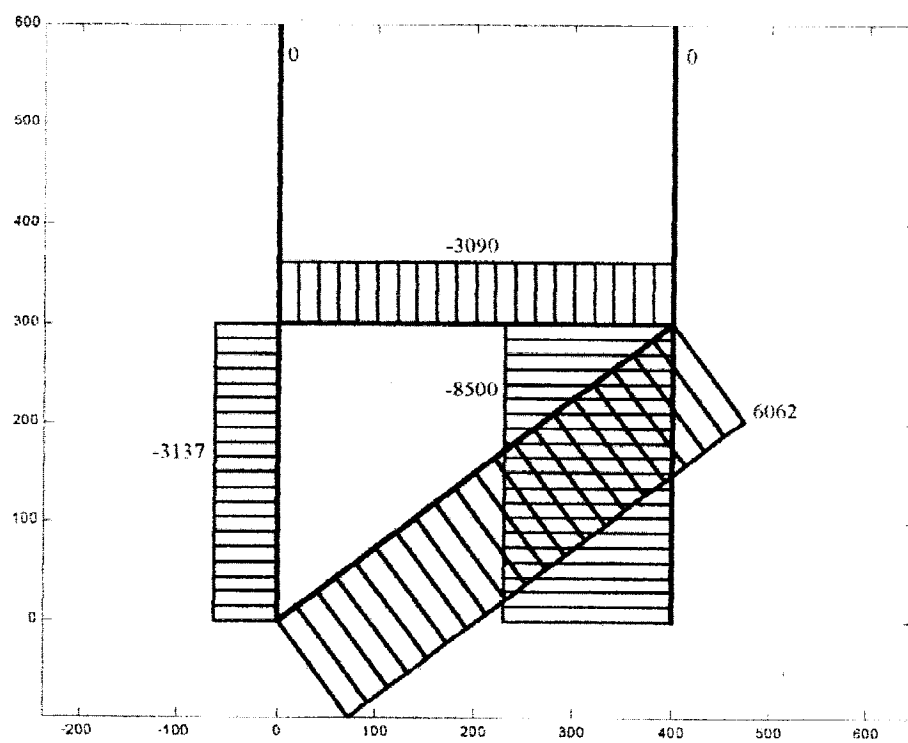




Punto B pag 1/2

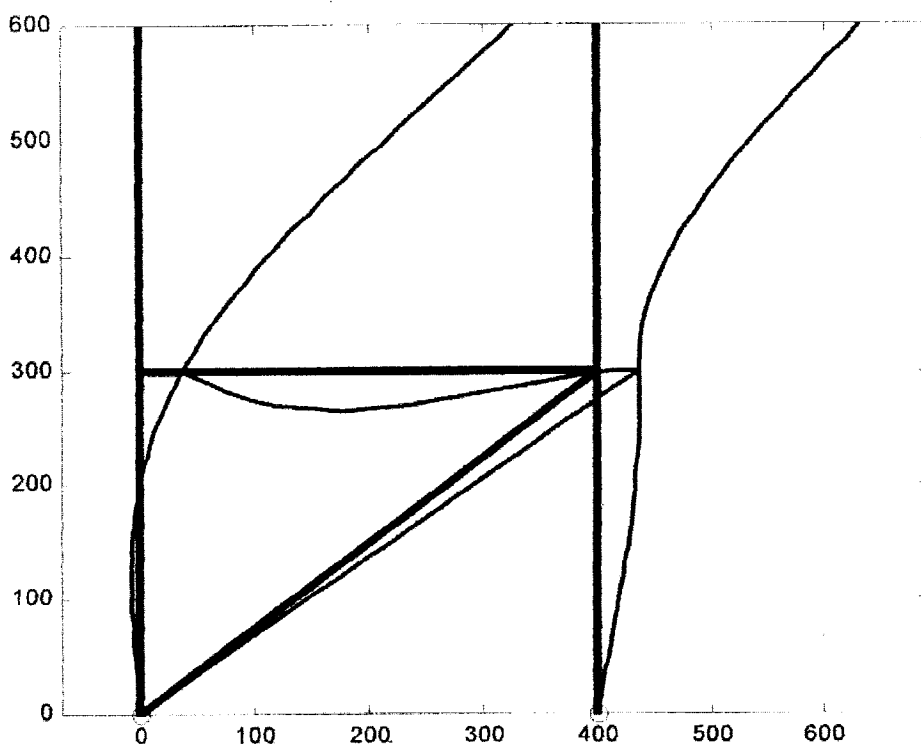


Punto B pag 2/2

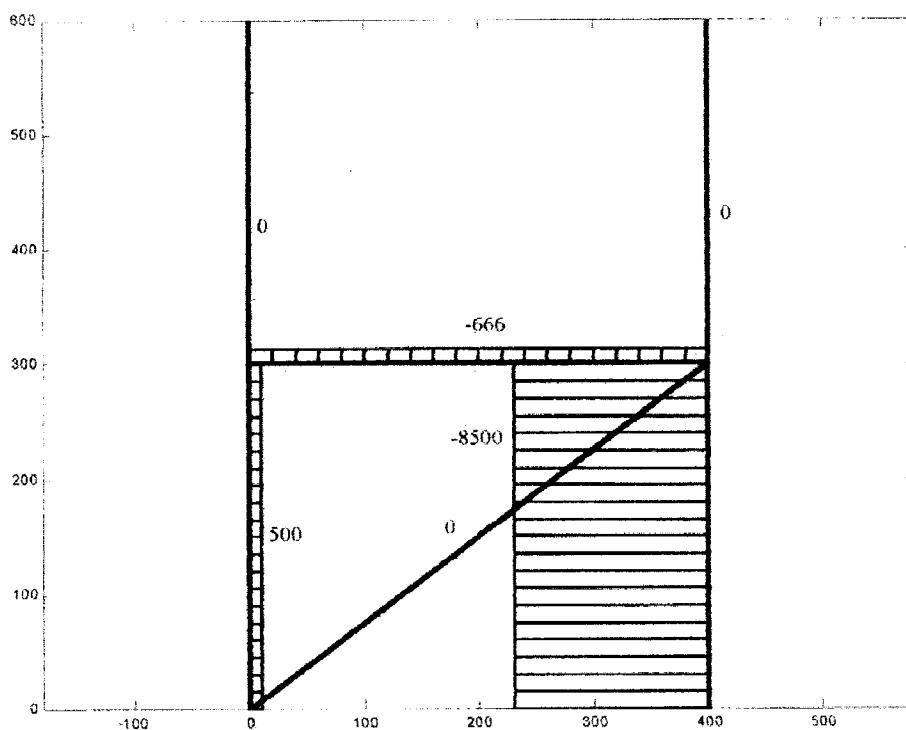


$N \text{ [Kg]}$

# Punti C e D pag 1/1

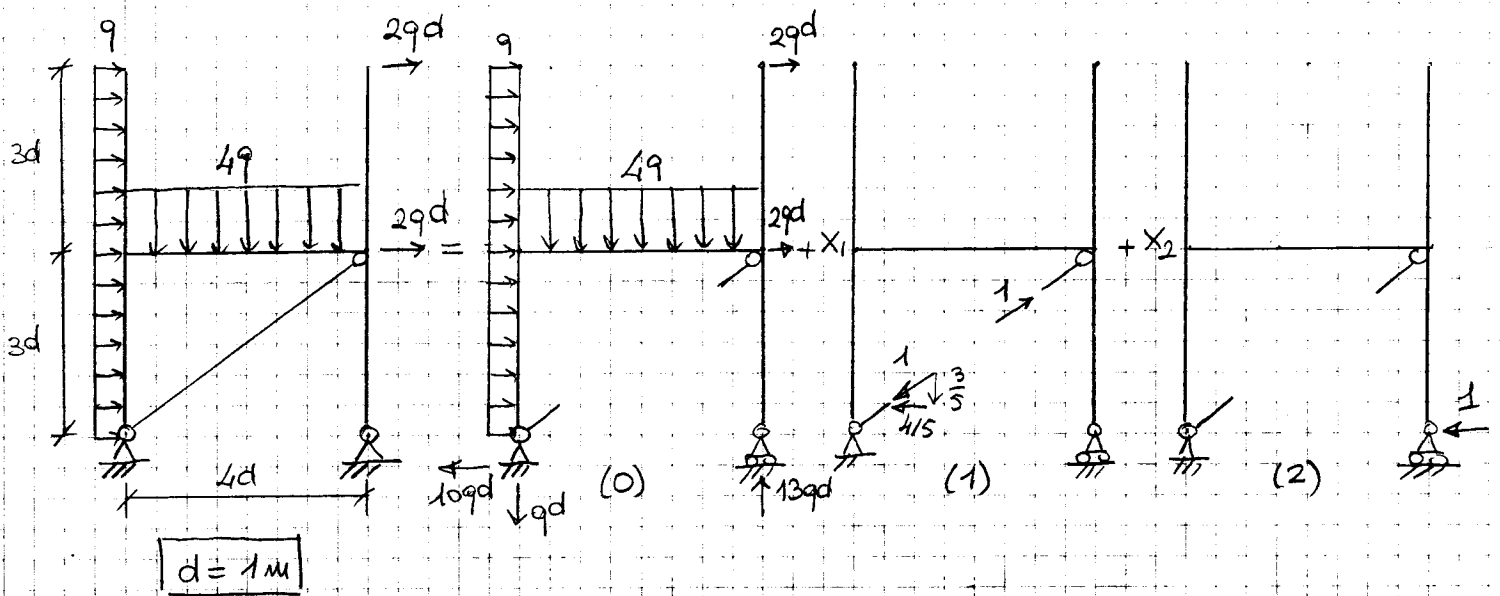


deformata



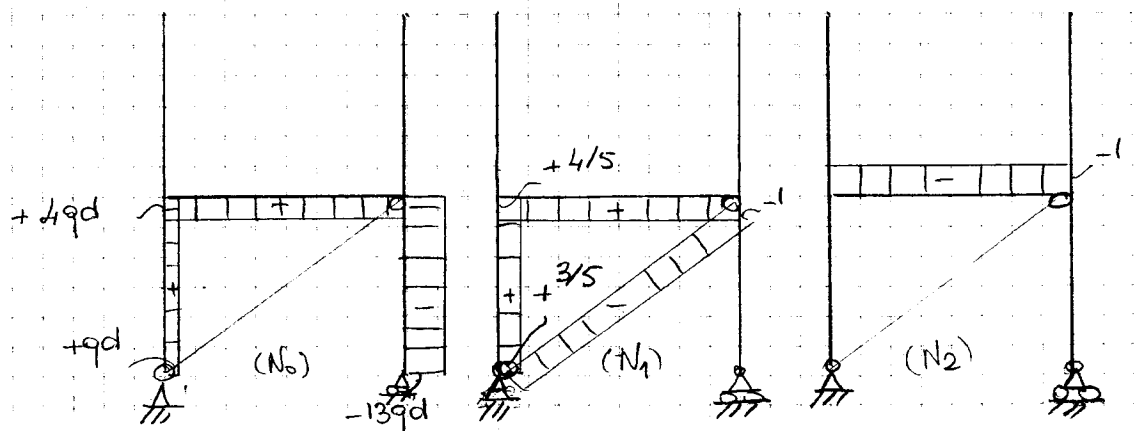
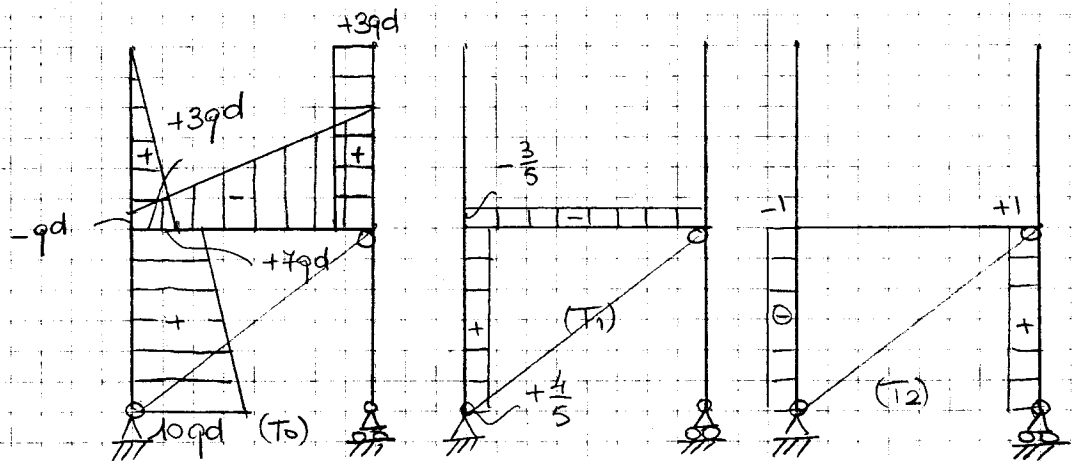
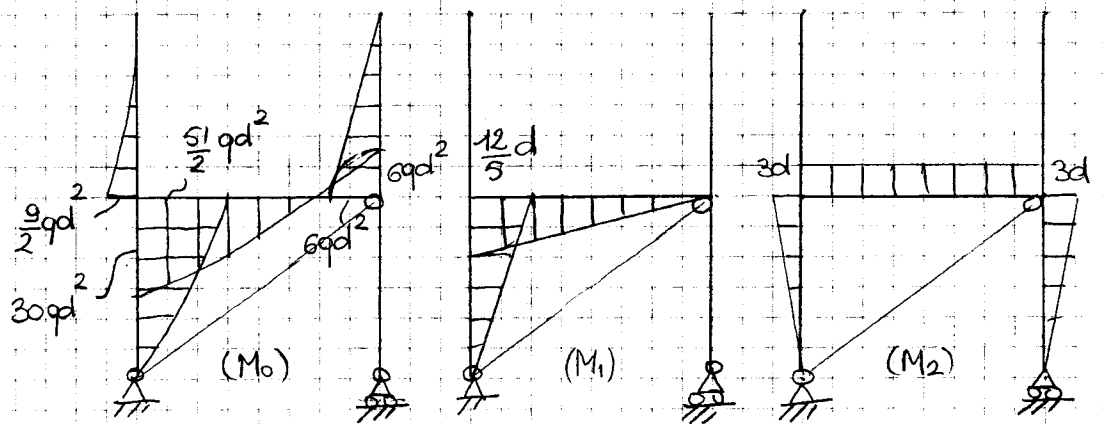
N [Kg]

# RISOLUZIONE CON IL METODO DELLE FORZE



$$\sin \alpha = \frac{3d}{5d} = \frac{3}{5}$$

$$\cos \alpha = \frac{4d}{5d} = \frac{4}{5}$$



$$\eta_{10} = \frac{1}{EJ} \left[ \int_0^{3d} (10qd x_3 - 9\frac{x_3^2}{2}) (\frac{4}{5}x_3) dx_3 + \int_0^{4d} (30qd^2 - qdx_3 - 49\frac{x_3^2}{2}) (\frac{12}{5}d - \frac{3}{5}x_3) dx_3 \right]$$

$$= \frac{1759}{10} \frac{qd^4}{EJ}$$

$$\eta_{20} = \frac{1}{EJ} \left[ \int_0^{3d} (10qd x_3 - 9\frac{x_3^2}{2}) (-x_3) dx_3 + \int_0^{4d} (30qd^2 - qdx_3 - 29x_3^2) (-3d) dx_3 \right]$$

$$= -\frac{2303}{8} \frac{qd^4}{EJ}$$

$$\eta_{11} = \frac{1}{EJ} \left[ \int_0^{3d} \left(\frac{4}{5}x_3\right)^2 dx_3 + \int_0^{4d} \left(\frac{3}{5}x_3\right)^2 dx_3 \right] = \frac{336 d^3}{25 EJ}$$

$$\eta_{12} = \frac{1}{EJ} \left[ \int_0^{3d} \left(\frac{4}{5}x_3\right) (-x_3) dx_3 + \int_0^{4d} (-3d) \left(\frac{3}{5}x_3\right) dx_3 \right] = -\frac{108 d^3}{5 EJ}$$

$$\eta_{22} = \frac{1}{EJ} \left[ 2 \int_0^{3d} (-x_3)^2 dx_3 + (-3d)^2 (4d) \right] = \frac{54 d^3}{EJ}$$

$$\begin{bmatrix} \eta_{11} & \eta_{12} \\ \eta_{12} & \eta_{22} \end{bmatrix} \begin{bmatrix} X_1 \\ X_2 \end{bmatrix} = \begin{bmatrix} -\eta_{10} \\ -\eta_{20} \end{bmatrix} \rightarrow \begin{cases} X_1 = -6328 \text{ kg} \\ X_2 = +134 \text{ kg} \end{cases}$$

Diagrammi dei momenti, progetto e verifica: vedi risoluzione con il metodo delle rigidezza